

Classification

Physics Abstracts

42.78 — 42.80 — 86.30R

## Ideal concentrators with polygonal absorbers (\*)

F. Bloisi, L. Vicari

Dipartimento di Fisica Nucleare, Struttura della Materia e Fisica Applicata,  
Facoltà di Ingegneria, Università di Napoli, 80125 Napoli, Italy

D. Ruggi

A.P.R.E. S.p.A., Roma, Italy

and F. Suraci

E.N.E.A.-F.A.R.E., Roma, Italy

(Reçu le 27 juillet 1984, révisé le 8 juillet 1985, accepté le 8 novembre 1985)

**Résumé.** — Nous avons introduit une nouvelle famille de concentrateurs stationnaires symétriques idéaux avec des absorbeurs polygonaux. Nous avons étudié en quelques détails leurs caractéristiques géométriques et nous avons montré que quelques-uns d'entre eux sont particulièrement aptes à se ranger dans des panneaux plans, et d'autres sont utiles lorsque l'isolement thermique est obtenu en insérant l'absorbeur dans un tube de verre sous vide.

**Abstract.** — We introduce a new family of ideal symmetrical stationary concentrators with absorbers having a polygonal cross section. We study in some detail their geometrical features and show that some of them are particularly well suited to be arranged in flat panels, while others are useful when the thermal insulation is obtained inserting the receiver into an evacuated glass pipe.

### Nomenclature

- $C$  Concentration factor
- $C^*$  Concentration factor for an ideal device
- $d$  Collector semiaperture (half of the entrance aperture width)
- $d'$  Absorber semiperimeter
- $h$  Maximum depth of the device
- $\theta_M$  Semiacceptance angle (half of the angular field of view)

An ideal cylindrical concentrator is a device able to collect radiation over an entrance aperture of width  $2d$ , an angular field of view of  $2\theta_M$  in the plane traverse to the cylindre and to concentrate all of it on an absorber of width  $2d'$ . The concentration factor  $C^* = d/d'$  is given by

$$C^* = 1/\sin \theta_M.$$

Ideal concentrators are often used to realize actual devices aimed to the collection of solar energy. In this case one has to consider other parameters besides the concentration factor, like the shape of the absorber when it has to be inserted into an evacuated glass tube or the depth of the mirror when it has to be assembled into a concentrating flat plate configuration.

In their paper [7] Winston and Hinterberger, developing an idea of Trombe and Meinel [8], introduced the cylindrical concentrator with absorbers of very general cross section. They dealt in more detail with circular, oval and fin absorbers. An example is given in figure 1a (Type 1 : circular absorber).

These concentrators use the whole surface of the

### 1. Introduction.

Since the first studies of Winston [1 to 3] and Baranov [4 to 6] on the Compound Parabolic Concentrator (CPC) many authors have dealt with the matter of new shapes for ideal concentrators [7 to 12].

(\*) Financially supported by E.N.E.A.-F.A.R.E.





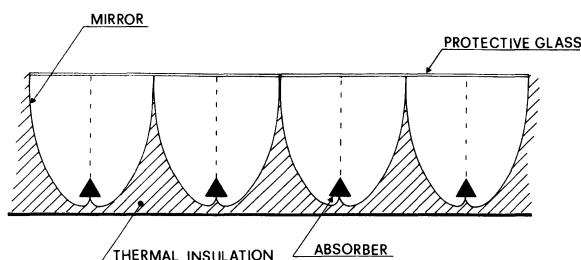


Fig. 5. — Concentrators assembled in a flat panel configuration.

For the Cartesian representation of the concentrator let us look at figure 4. The equation of the arc AB is that of a circumference and depends on the choice for L. The arc BD' has equation

$$x^2 = 4 y d' .$$

If we look for the point D' ≡ (x<sub>D'</sub>, y<sub>D'</sub>), as previously done, we find :

$$x_{D'} = 2 d' / \tan \theta_M \quad y_{D'} = d' / \tan^2 \theta_M .$$

From geometrical considerations on figure 6 we get also

$$2 OD' = x_{D'} / \cos \theta_M = 2 d' / \sin \theta_M .$$

The concentration factor is then given by :

$$C = d/d' = OD'/d' = 1/\sin \theta_M = C^*$$

and therefore each concentrator obtained in this way is ideal.

The depth *h* of the concentrators of this family is given by

$$h = d \tan \theta_M + AL (1 - \cos \theta_M) + d \sin \theta_M \cos \theta_M$$

and, for given semiaperture *d* and semiacceptance angle  $\theta_M$ , depends on the choice for the point L. The value of AL which minimize the device's depth is given by the existence condition of L mentioned above. In this case the triangle CAL is rectangular in A, the receiver is a triangle, and we can write

$$AL = d \sin^2 \theta_M / (1 + \sin \theta_M)$$

$$CL = d \sin \theta_M / (1 + \sin \theta_M) .$$

The resulting concentrator is shown in figure 1c (Type 3 : delta absorber).

Such a concentrator can be very useful for applications in flat panels with external cover glass. Nevertheless, just like the concentrator with circular receiver, it is useless if the insulation is obtained by means of cylindrical vacuum pipe.

In figure 1d we show a device (Type 4 : inscriptible quadrilateral absorber) which has both the advantages of small depth and negligible optical losses. Its receiver is obtained as before, but now the triangle CLA is rectangular in L. The resulting absorber is a quadri-

later inscriptible in a circumference. In this case we have

$$AL = d \sin^2 \theta_M / (\sin \theta_M + \cos \theta_M)$$

$$CL = d \sin \theta_M \cos \theta_M / (\sin \theta_M + \cos \theta_M) .$$

#### 4. Conclusions.

There are two main ways to assemble the solar concentrators for heating purposes.

In the first case (flat panels, Fig. 5), many concentrators are inserted into a box ; the top is closed by a single or double glass and the bottom is thermally insulated.

In the second case (Fig. 1d), as mentioned above, the thermal insulation is obtained by inserting the receiver into an evacuated glass pipe.

We discussed here five different geometries of concentrators. All of them may be used in the flat panel configuration while only the types 2 and 4 may be used for glass pipes. The use of the types 1 and 3 in this configuration imply an arbitrary cut of the mirror or an arbitrary shift of the receiver, or both, resulting in a non ideal concentrator with a lot of optical losses, the amount of which depends on the shape of the actual device.

In both configurations all of the types 1, 2, 3 and 4 are equivalent with respect to the problem of the thermal insulation. Only the CPC has the disadvantage of a thermal wasting area which is twice the absorbing one ; furthermore half of it is in direct contact with the bottom of the box.

Table I. — Depth to semiaperture ratio versus the semi-acceptance angle.

Semi acceptance $\theta_M$	Depth/semiaperture <i>h/d</i>				
	CPC	Type 1	Type 2	Type 3	Type 4
0°	∞	∞	∞	∞	∞
10°	6.537	6.076	6.115	5.843	5.843
20°	3.687	3.237	3.289	3.074	3.074
30°	2.598	2.300	2.354	2.187	2.190
40°	1.958	1.831	1.876	1.743	1.753
50°	1.482	1.540	1.564	1.450	1.480
60°	1.077	1.329	1.321	1.211	1.285
70°	0.706	1.152	1.104	0.985	1.139
80°	0.350	0.987	0.891	0.751	1.039
90°	0.000	0.818	0.667	0.500	1.000

In the following we show the expressions of the ratio between the depth and the semiaperture for all the types of concentrators (see Table I) :

$$\text{CPC} : h/d = \cotg \theta_M + \cos \theta_M$$

$$\text{Type 1} : h/d = \cotg \theta_M + 1/\pi + (\sin \theta_M)/2$$

$$\text{Type 2} : h/d = \cotg \theta_M + (\cos \theta_M + 2 \sin \theta_M)/3$$

$$\text{Type 3} : h/d = \cotg \theta_M + \\ + \sin \theta_M (\sin \theta_M + \cos \theta_M) / (1 + \sin \theta_M)$$

$$\text{Type 4} : h/d = c \tan \theta_M + \\ + (\sin^2 \theta_M + \sin \theta_M \cos^2 \theta_M) / (\sin \theta_M + \cos \theta_M)$$

In the flat panel configuration, for  $\theta_M \leq 50^\circ$  type 3 has global advantages over the other ones, while for  $\theta_M > 60^\circ$  the choice between the CPC and type 3 depends on the relative importance of space, mirror and insulation costs.

In the glass pipe configuration, for  $\theta_M \leq 60^\circ$  type 4 has global advantages over type 2, while this seems better for  $\theta_M > 60^\circ$ .

#### References

- [1] WINSTON, R., *J. Opt. Soc. Am.* **60** (1970) 245.  
 [2] HINTERBERGER, H. and WINSTON, R., *Rev. Sci. Instrum.* **37** (1966) 1094.  
 [3] WINSTON, R., *Solar Energy* **16** (1974) 89.  
 [4] BARANOV, V. K., *Geliotekhnika* **2** (1966) 11.  
 [5] BARANOV, V. K., *Geliotekhnika* **2** (1975) 36.  
 [6] BARANOV, V. K., *Opt. Mekhnic. Promyshlennost*, **6** (1965).  
 [7] WINSTON, H. and HINTERBERGER, H., *Solar Energy* **17** (1975) 255.  
 [8] Patent by TROMBE, F., MEINEL, A. B. *et al.* presented by McKenney D. B. at the National Science Foundation Solar Thermal Review (March 1974).  
 MEINEL, A. B. and MEINEL, M. P., *Applied Solar Energy : an introduction* (Addison Wesley, Reading, Mass.) 1976.  
 [9] RABL, A., *Solar Energy* **18** (1976) 93.  
 [10] MILLS, D. R. and GIUTRONICH, J. E., *Solar Energy*, **25** (1979) 85.  
 [11] MCINTIRE, W. R., *Solar Energy* **25** (1980) 215.  
 [12] GUAY, E. J., *Solar Energy* **24** (1979) 265.